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Design, optimization and fabrication of surface micromachined pressure sensors

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Abstract

Surface micromachined pressure sensors have been designed, optimized and fabricated. Polysilicon diaphragms with thickness of 2 μ m have been manufactured by means of LPCVD (Low Pressure Chemical Vapor Deposition) and used as the core elements in micro pressure sensors. The diaphragms stand on top of micro vacuum cavities that are constructed by a specially designed, vacuum sealing process [1]. Pressure changes cause the diaphragms to deflect. The magnitudes of pressure are detected by using built-in piezoresistive resistors that are placed on the surface of the thin diaphragms. Both square- and circular-shape diaphragm pressure sensors with nominal widths (diameters) of 100 μ m have been fabricated and tested. Optimized designs, including position, orientation and length of the piezoresistive resistors, have been analyzed. The fabrication process is fully compatible with IC (Integrated Circuit) processing. Therefore, micro electronics can be integrated directly with these micro sensors and microscale mechatronics systems can be realized. Prototype pressure sensors which are designed for 100 Psi in full operational scale have a measured sensitivity of 0.15 mV/V/Psi and a maximum linearity error of $\pm 0.1\%$ FSS (Full Scale Span). © 1998 Elsevier Science Ltd. All rights reserved.

1. Introduction

The emerging area of MEMS (Microelectromechanical Systems) has originated from IC processing. After decades of research and development, the state of art MEMS processes are capable of integrating microelectronics and sensors on a single

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chip [2, 3]. The combination of microelectronics and mechanical components makes MEMS more powerful and versatile than the conventional sensors. The possibility of applying theories and practice in the macro scale mechatronics systems to the microelectromechanical miniaturized systems is both attractive and challenging. This paper presents a new type of micro pressure sensor that is aimed for small size, light weight and low cost. Micro pressure sensor is the most widely used MEMS product in the market and is an excellent example of microscale mechatronics system.

Bulk-micromachined pressure sensors are one of the earliest products made by silicon micromachining [4]. These first generation MEMS pressure sensors were developed in the 1970s. Today, many companies fabricate and sell bulk-micromachined pressure sensors for automobile, industrial and biomedical applications. The measurement range for these pressure sensors can be as high as 10,000 Psi with excellent reliability. For example, Foxboro Company has reported that their pressure sensors can sustain 5 billion 0–10,000 Psi pressure cycles. Since these bulk-micromachined pressure sensors have been investigated for many years, the knowledge in both areas of fabrication and design is abundant [5–9].

On the other hand, pressure sensors based on surface micromachined diaphragms were first suggested and fabricated in the 1980s [10]. Thin film deposition and reactive sealing technologies were used to fabricate polysilicon diaphragms with cavities underneath. The back side silicon wet etching process which has been used for bulkmicromachined pressure sensors was avoided [11-14]. These surface micromachined pressure sensors may be more attractive than the bulk-micromachined ones because of the following reasons. First, bulk-micromachined pressure sensors require anisotropic silicon etching [15] to create thin diaphragms from the back side of silicon wafers. This process consumes large areas [16]. For example, if a standard four inch wafer with thickness of 500 μ m is used, an area of about $800 \times 800 \ \mu \text{m}^2$ is required to make a $100 \times 100 \ \mu\text{m}^2$ diaphragm. However, an area of only $100 \times 100 \ \mu\text{m}^2$ is needed to make a surface-micromachined diaphragm. Second, bulk-micromachined pressure sensors require post processing including glass to silicon bonding before the final packaging process. Surface micromachined pressure sensors are ready for packaging after the micromachining processes. Finally, surface micromachining is easier to be integrated with IC processes for additional signal processing or device functionality.

This paper presents a new fabrication process for surface micromachined pressure sensors. Design issues for surface micromachined diaphragms are investigated. This new process uses an LPCVD polysilicon deposition step to seal the vacuum cavity [1]. Design optimizations for piezoresistive sensing resistors including position, orientation and length will be addressed.

2. Operation principle and manufacturing method

2.1. Operational principle

When a uniform pressure is applied to a silicon micro diaphragm, deflection occurs and the internal strain of the diaphragm changes. Silicon is a piezoresistive material [17] such that its resistance changes when the internal strain varies. If the pressure sensing resistors can be constructed and placed on top of the thin diaphragm, pressure can be measured by monitoring the resistance changes. In this paper a Wheatstone bridge type circuitry has been used to give voltage outputs for pressure measurement. Two sensing resistors are placed inside the diaphragm and the other two reference resistors are placed outside the diaphragm.

2.2. Small deflection theory

In order to determine the optimal design of the sensing resistors including position, orientation and length, conventional theories of solid mechanics have been investigated. Since linear response is desired, these pressure sensors are designed to be operated in the small deflection region. Two types of diaphragms are considered including both circular and square shapes. They are discussed and analyzed separately.

2.2.1. Circular plates under small deflection

When a circular plate is under a uniform pressure, q, the governing equation can be derived in terms of the cylindrical coordinate system. The deflection function is derived as [17]:

$$w = \frac{qr^4}{64D} + \frac{C_1r^2}{4} + C_2\log\frac{r}{a} + C_3 \tag{1}$$

Where r is the radial coordinate in the cylindrical coordinate system; a is the radius; D is the flexural rigidity of the diaphragm; C_1 , C_2 , C_3 are the constants to be determined by the boundary conditions of the diaphragm. For micro diaphragm with clamped boundary, the radial and circumferential strain can be derived from eqn (1) as:

$$\varepsilon_r = -\frac{3qa^2(1-v^2)}{8Eh^2} \left(1 - 3\left(\frac{r}{a}\right)^2\right) \tag{2}$$

$$\varepsilon_0 = -\frac{3qa^2(1-v^2)}{8Eh^2} \left(1 - \left(\frac{r}{a}\right)^2\right)$$
 (3)

E is the Young's modulus; v is Poisson's ratio and h is the thickness of the diaphragm, respectively. Based on these formulas, Fig. 1 shows the strain distributions of a circular diaphragm (100 μ m in diameter, 2 μ m in thickness) which is under a uniform pressure of 100 Psi. Only half of the diaphragm is plotted due to symmetry. It is observed that the radial strain goes from the compressive state (at the center) to the tensile state (at the edge). The circumferential strain is all the way compressive and the maximum value is at the center of the diaphragm. It can be concluded from Fig.

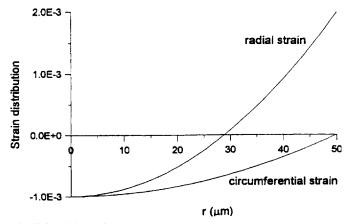


Fig. 1. Radial and circumferential strain distributions of a circular shape diaphragm.

1 that radial strain at the edge of the diaphragm has a higher value than other positions.

2.2.1. Square plates under small deflection

For square shape plate under small deflection, the analytical solution is more complicated. Timoshenko's method [19] has been used in this paper. A more general case for rectangular shape diaphragms is derived here. It assumes that the deflection of a rectangular shape plate with clamped edges is the summation of three deflection components: w_1 , w_2 and w_3 , w_1 is the deflection of a simply supported plate under the same uniform pressure; w_2 and w_3 are introduced to preserve the clamped boundary conditions. When a rectangular shape plate with width, a, b and thickness, b, is under a uniform pressure, w_1 , w_2 and w_3 can be solved respectively as:

$$w_{1} = \frac{4qa^{4}}{\pi^{5}D} \sum_{m=\text{odd}} \left(\frac{1}{m^{5}}\right) (-1)^{\left(\frac{m-1}{2}\right)} \cos\frac{m\pi x}{a}$$

$$\times \left(1 - (A_{m} \tanh A_{m} + 2) \left(\frac{\cosh\frac{m\pi y}{a}}{2\cosh A_{m}}\right) + \frac{m\pi y}{a} \sinh\frac{m\pi y}{a}\right) \quad (4)$$

$$w_{2} = \frac{-a^{2}}{2\pi^{2}D} \sum_{m=\text{odd}} E_{m} (-1)^{\left(\frac{m-1}{2}\right)} \left(\frac{1}{m^{2} \cosh A_{m}} \cos\frac{m\pi x}{a}\right)$$

$$\times \left(\frac{m\pi y}{a} \sinh\frac{m\pi y}{a} - A_{m} \tanh A_{m} \cosh\frac{m\pi y}{a}\right) \quad (5)$$

$$w_{3} = \frac{-b^{2}}{2\pi^{2}D} \sum_{m=\text{odd}} F_{m}(-1)^{\left(\frac{m-1}{2}\right)} \left(\frac{1}{m^{2}\cosh B_{m}}\cos\frac{m\pi y}{b}\right) \times \left(\frac{m\pi x}{b}\sinh\frac{m\pi x}{b} - B_{m}\tanh B_{m}\cosh\frac{m\pi x}{b}\right)$$
(6)

where

$$A_m = \frac{m\pi b}{2a}, \quad B_m = \frac{m\pi a}{2b} \tag{7}$$

The values of E_m and F_m can be simulated by the following equations:

$$\frac{4qa^2}{\pi^3 n^4} \left(\frac{A_n}{\cosh^2 A_n}\right) - \tanh A_n = \frac{E_n}{n} \left(\tanh A_n + \frac{A_n}{\cosh^2 A_n}\right) + \frac{8na}{\pi b} \sum_{m = \text{odd}} F_m \frac{1}{m^3 \left(\left(\frac{a}{b}\right)^2 + \left(\frac{n}{m}\right)^2\right)^2} \tag{8}$$

$$\frac{4qb^2}{\pi^3 n^4} \left(\frac{B_n}{\cosh^2 B_n}\right) - \tanh B_n = \frac{F_n}{n} \left(\tanh B_n + \frac{B_n}{\cosh^2 B_n}\right) + \frac{8nb}{\pi a} \sum_{m = \text{odd}} E_m \frac{1}{m^3 \left(\left(\frac{a}{b}\right)^2 + \left(\frac{n}{m}\right)^2\right)^2} \tag{9}$$

The strain distribution can then be derived by using the fundamental strain-displacement relation.

$$\varepsilon_{xx} = -z \frac{\partial^2 w}{\partial x^2}, \quad \varepsilon_{yy} = -z \frac{\partial^2 w}{\partial y^2}, \quad \varepsilon_{xy} = -2z \frac{\partial^2 w}{\partial x \partial y}$$
(10)

Figure 2 shows the analytical x-directional strain distributions of a square shape diaphragm (100 μ m in length and 2 μ m in width) which is under a pressure of 100 Psi. This figure only shows the strain distributions at the right top corner of the diaphragm due to symmetry. The y-directional strain distribution is symmetrical to the x-directional strain with respect to the line of x = y. It is observed that the maximum strain occurs at the middle edge of the square shape diaphragm where the strain is in tensile state. The strain decreases and becomes compressive at the center of the plate.

2.3. Sensitivity of piezoresistivity

Piezoresistivity of silicon can be analyzed in terms of two directions: parallel or perpendicular direction to the electrical current. The relation can be derived as:

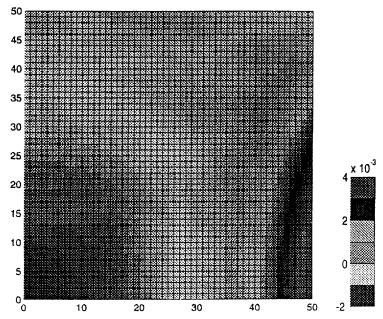


Fig. 2. The x-directional strain distributions on a 100 μ m square shape diaphragm (right top quarter) under a pressure of 100 Psi.

$$\frac{\Delta \rho}{\rho} = F_{\text{par}} \varepsilon_l + F_{\text{per}} \varepsilon_t \tag{11}$$

The symbol ε_t and ε_t are the strain parallel and perpendicular to the direction of the electrical current, respectively. $F_{\rm par}$ and $F_{\rm per}$ are the gauge factors. They can be represented with respect to the piezoresistive constant Π as:

$$F_{\text{par}} = \frac{E(\Pi_l + \nu \Pi_t)}{(1 - \nu^2)} \tag{12}$$

$$F_{\text{per}} = \frac{E(\Pi_i + \nu \Pi_l)}{(1 - \nu^2)} \tag{13}$$

and the changes of resistance can be derived as:

$$\frac{\Delta R}{R} = F_{\text{par}} \varepsilon_l + F_{\text{per}} \varepsilon_t + \varepsilon_t - \varepsilon_l - \varepsilon_z \tag{14}$$

which can be further derived as:

$$\frac{\Delta R}{R} = \left(F_{\text{par}} + \frac{1}{1 - \nu}\right) \varepsilon_l + \left(F_{\text{per}} + \frac{1 - 2\nu}{1 - \nu}\right) \varepsilon_t \tag{15}$$

The sensitivity of either square or circular shape diaphragms can be calculated according to the above analytical derivation.

2.4. Optimization of resistor length

After the sensitivity distribution of the piezoresistive resistor is calculated, the optimal positions for sensing resistors are obtained. The next step is to determine the geometry of the resistor including length and shape for optimal sensitivity. In order to maximize the piezoresistivity effects, line shape resistors are designed for detecting strain changes parallel to the direction of the input current. These line shape resistors are placed at the edges of either the circular or the square shape diaphragm to gain maximum sensitivity as discussed in the previous section. Since misalignments are unavoidable during the fabrication processes, the sensing resistors are designed to have a non-effective portion of 2 μ m outside the diaphragm [20]. In order to match the desired impedance and to fully utilize the most sensitive regions on the diaphragms, resistors with several turns are designed in some cases. The turning portions of the resistor will be covered by metal in the manufacturing process such that they are considered as the non-effective parts. Only the effective parts of the sensor that are inside the diaphragm will be active during pressure measurements. According to the Wheatstone bridge configuration, the output voltage is derived as:

$$V_{\text{out}} = \frac{\Delta R_{\text{eff}}}{2(R_{\text{eff}} + NR) + \Delta R_{\text{eff}}} V_{\text{in}}$$
(16)

where NR is the magnitude of non-effective resistance, $R_{\rm eff}$ is the value of the effective resistance and $\Delta R_{\rm eff}$ is the change of the effective resistance due to outside pressure. It can be derived as:

$$\frac{\Delta R_{\text{eff}}}{R_{\text{eff}}} = \frac{1}{l} \int_{L}^{L-l} \frac{\Delta R}{R} \, \mathrm{d}x \tag{17}$$

Where l is the length of the effective resistor, L is the characteristic length of the diaphragm (radius for a circular shape diaphragm and half of the width for a square shape diaphragm).

The optimal length of the effective resistor can be calculated by using the above derivation. Figure 3 shows the simulation result for a 100 μ m in diameter, 2 μ m in thickness, circular shape diaphragm. It is observed that when the effective resistor length is about 12 μ m, the diaphragm has an optimal output. Increasing or decreasing this length will reduce the output voltage.

Figure 4 shows the simulation results for square shape diaphragms. The simulation is based on a 100 μ m in length, 2 μ m in thickness, square shape diaphragm which is under a pressure of 100 Psi. The optimal effective length is about 10 μ m in this case. The standard resistor design has one single turn. The sheet resistance is 100 ohm and the width of the resistor is 2 μ m. It is observed in Fig. 4 that increasing the number of the turns or the sheet resistance will increase the sensitivity. Increasing the width of the resistor or the thickness of the diaphragm will decrease the sensor output.

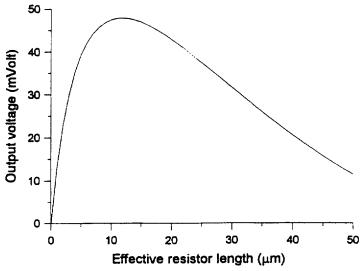


Fig. 3. Optimization analysis of effective resistor length on a circular shape diaphragm.

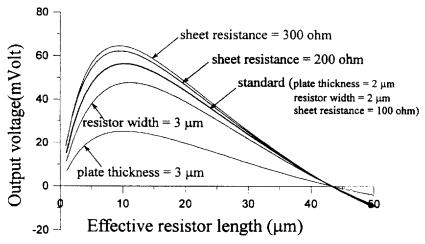


Fig. 4. Optimization analysis of effective resistor length on a square shape diaphragm.

3. Fabrication processes and results

The fabrication of the sensor requires a total of six masks. There have been many fabrication obstacles to be overcome in the facility we have used in Taiwan. First, the capability of dry etching was not well characterized in the microelectronics laboratory for our special processes. These processes include the dry etching of thick polysilicon layer and silicon. Therefore, an all wet etching process has been designed and

developed to overcome these problems. Second, the LPCVD oxide deposition furnace was not available during our process, a special process involving LPCVD polysilicon deposition and oxidation has been used to replace the LPCVD oxide deposition process.

The process starts with a silicon wafer that is first cleaned. A layer of 0.25 μ m LPCVD silicon nitride is deposited and a layer of 0.2 μ m thick LPCVD polysilicon is deposited afterwards. The polysilicon layer is the mask for the wet etching process of silicon nitride. The shape of the cavity is defined by using the first mask (mask No. 1) and polysilicon is etched by using a wet silicon etching solution of NHO₃: $H_2O: NH_4F = 64:33:3$ for about one and a half minutes. Silicon nitride is then removed by using polysilicon as the masking material in H_3PO_4 liquid at $175^{\circ}C$ for about 1 hour. The cavity area is then formed by the wet silicon etching solution for about 2 min. During the same time, the polysilicon layer on top of the silicon nitride is also removed. The depth of the cavity is about 0.5 μ m. At the end of the step, a cavity is formed. The wafer is then sent to the thermal oxidation furnace to grow about 1 μ m thick silicon dioxide which fully covers the cavity to the original height. After these steps, Fig. 5(a) applies.

A layer of $0.25 \,\mu\text{m}$ polysilicon is then deposited and oxidized to form a layer of $0.5 \,\mu\text{m}$ thick silicon dioxide. The second mask is then used to define the etch channel shape on the silicon dioxide layer. Buffered HF has been used in this wet etching process for about 5 min. A layer of $0.8 \,\mu\text{m}$ undoped LPCVD polysilicon is then deposited as the first layer of the diaphragm. The third mask is used to define the etch holes on polysilicon by using wet silicon etchant. Figure 5(b) is the cross sectional view of the process flow at this stage.

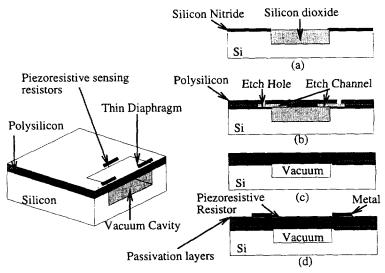


Fig. 5. Process sequence for the surface micromachined pressure sensors.

The etch hole is in direct contact with the etch channel which is in contact with the cavity. Concentrated HF can etch away the etching channel as well as the cavity which is also made of silicon dioxide. During this etching process, the etch front can be observed under an optical microscope. When the etch front gradually merged and disappeared at the center of the diaphragm, it indicates the completion of the cavity etching. A 1.5 μ m thick layer of LPCVD polysilicon is deposited after the cavity etching. This thick polysilicon layer will seal the etch channel since the etch channel is only 0.5 μ m in thickness. The sealing process is carried out at a low vacuum environment (LPCVD deposition) such that the cavity is at low vacuum condition at the end of the sealing process. Figure 5(c) is the process flow diagram up to this step.

After the cavity is sealed and the polysilicon diaphragm is formed, a layer of 0.1 μ m thick nitride is deposited. A layer of 0.1 μ m thick polysilicon is deposited and then thermal oxidized to form a silicon dioxide layer. Both the silicon dioxide and silicon nitride layers function as the masking layers for the definition of the piezoresistive sensing resistors. The forth mask is used to define the piezoresistive sensors. First, wet etching of the silicon dioxide in buffered HF is conducted to define the shape of the piezoresistive resistors. The nitride masking layer is then etched by H_3PO_4 liquid at 175 °C for about 1 hour. The shape of the polysilicon resistors is now defined and the wafer is put into the phosphorous pre-deposition furnace for the phosphorus drive-in process for about an hour. This process will grow about 0.1 μ m thick thermal oxide which is used as the passivation layer for the device.

The fifth mask is now used to define the contact holes by wet etching the oxide layer. Finally, a layer of aluminum is deposited onto the wafer and the sixth mask is used to define the interconnection lines and contact pads. Wet etching of aluminum is performed by using a solution of H_3PO_4 : HNO_3 : CH_3COOH : $H_2O = 50:2:10:9$ at $60^{\circ}C$ for about 25 seconds. Figure 5(d) shows the completion of these steps. Figure 6 shows top view of a completed surface micromachined pressure sensor with a square shape diaphragm.

Circular shape pressure sensors have also been made by using the same process. Figure 7 is an SEM microphoto of a fabricated circular shape pressure sensor. It has been cleaved such that the cavity underneath the diaphragm can be clearly observed. Metal lines, contact pads, piezoresistive resistors, etch holes and etch channels can also be seen in Fig. 7.

We have measured the performance of devices made of thin polysilicon diaphragms with the same geometry as shown in Fig. 8 [19]. These MEMS pressure sensors have measured sensitivity of $0.15 \, \text{mV/V/Psi}$ and a maximum linearity error of $\pm 0.1\%$ FSS (Full Scale Span) at room temperature. The sensitivity increases as the temperature decreases because the piezoresistivity changes with temperature [21]. However, the linearity remains excellent at different temperatures as shown in Fig. 8. The temperature range of -40 to 120°C fits well for the aerospace applications (-60°C is the desired range). These pressure sensors have also demonstrated good performance in long term drift, repeatability, pressure hysteresis and temperature hysteresis. We are still in the process of testing these pressure sensors for qualifications of aerospace standards.

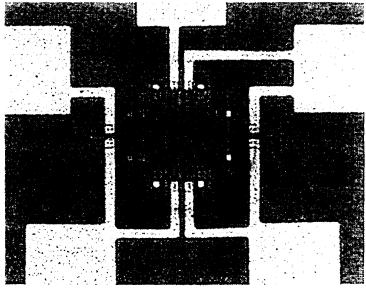


Fig. 6. A fabricated surface micromachined pressure sensor with a square shape diaphragm under an optical microscope.

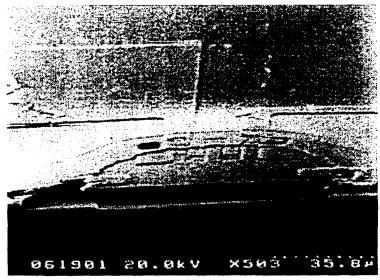


Fig. 7. A SEM micrograph showing a cleaved circular micro shell. The cavity underneath the diaphragm can be seen clearly.

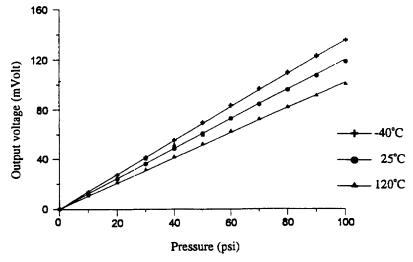


Fig. 8. Testing results of MEMS pressure sensors at different temperatures.

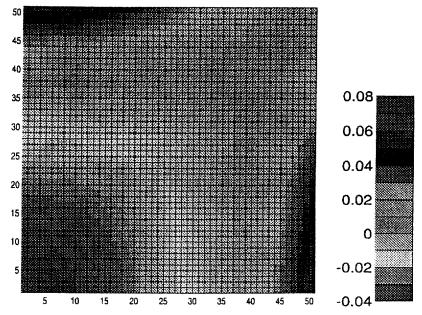


Fig. 9. Sensitivity distributions of a resistor placed 45° off the x-axis on a square diaphragm.

4. Discussions

Several designs and fabrication issues are discussed for surface micromachined pressure sensors. As discussed in the design principles for pressure sensors, the piezo-

resistive sensing resistors should be placed at the most sensitive places on the diaphragm. In the pressure sensor design by Motorola, X-ducers [2], where resistors are placed 45° off the axial direction on a square shape diaphragm. To determine the effects of orientation on the placement of these resistors, a sensitivity check for different orientations has been investigated. In a square shape diaphragm, if the resistor is placed with an angle, ϕ , with respect to the x-axial direction, the strains can be derived as:

$$\varepsilon_l = \varepsilon_x \cos^2 \phi + 2\varepsilon_{xy} \cos \phi \sin \phi + \varepsilon_y \sin^2 \phi \tag{18}$$

$$\varepsilon_t = \varepsilon_r \sin^2 \phi - 2\varepsilon_{rr} \cos \phi \sin \phi + \varepsilon_r \cos^2 \phi \tag{19}$$

The sensitivities for different orientations of resistor placement can then be calculated. Figure 8 shows the simulation result of the resistance sensitivity diagram for a resistor placed in a direction 45° off the x axis. The sensitivity distribution is, as expected, symmetrical to the line of x = y. The overall sensitivity is comparable to the orientation that is parallel to the x axis as previously discussed but the most sensitive area seems to be decreased.

Other than the orientation, issues including the size and electrical properties of the resistors should also be considered. Some of these parameters have been investigated as shown in Fig. 4. Increasing the turns and sheet resistance are two ways to increase the sensor output. On the other hand, increasing the width of the sensing resistor will reduce the overall output. The depth of the resistor also affects the performance. The top and bottom surfaces have the largest strain changes and the central plane of the diaphragm has zero strain changes according to classical theories of mechanics. It is recommended that the piezoresistive resistors should be constructed close to the surface of the diaphragm. In other words, the drive-in diffusion procedure should be as shallow as possible. Gauge factors of the piezoresistivity are determined by the property of the thin films which can vary from different process runs. These factors should be characterized and tuned to the optimal values before the pressure sensors are made. Finally, the process developed here has minimized the residual stress by using several annealing processes. However, it is expected that tiny residual stress may exist and the effect of residual stress to the overall performance of the pressure sensors is still under investigation.

The fabrication process developed in this paper has been designed to overcome some process obstacles. A similar process can be developed if dry etching capabilities are well characterized and LPCVD furnaces including silicon dioxide and low stress silicon nitride are available [20]. Ion implantation for defining piezoresistive resistors is also recommended for future development. In that case, photoresist should be used during the ion implantation and the passivation layer should be deposited afterwards.

5. Conclusions

Design, modeling and fabrication of surface micromachined pressure sensors have been successfully established. The orientation, position and length of the sensing resistors have been fully analyzed. Several specific conclusions can be drawn for the sensor geometries discussed in this paper. For circular shape diaphragms, the most sensitive positions are found to be at the edge of the diaphragm. The sensing resistors should be placed in the radial directions and the optimal length is 12 μ m. For square shape diaphragm, the most sensitive locations are at the middle edge of the diaphragm. The sensing resistors should be oriented toward the center and the optimal length is 10 μm. These optimal designs have been analyzed based on conventional solid mechanics theories of shell and plate. Instead of using numerical analysis, analytical solutions are derived. These analytical solutions provide important design guidelines when the dimensions, shapes or orientations of the diaphragm or sensing resistors are changed. An all wet etching process has been developed to fabricate the surface micromachined pressure sensors. These steps include wet etching of silicon dioxide, silicon nitride, silicon; thin film deposition of nitride, polysilicon; oxidation and drive-in processes. All of these process parameters have been successfully established. These parameters are suitable not only for fabricating the surface micromachined pressure sensors but also for fabricating other MEMS devices.

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